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(54) Hydraulically damped mounting device

(57) A hydraulically damped mounting device has two anchor parts (1,4) connected by a resilient wall (23) which bounds a working chamber (8) filled with hydraulic fluid. Radial side chambers (28, 31) also filled with hydraulic fluid are formed in the resilient wall and connected to each other and to the working chamber by a connecting channel (26, 29). The existence of the radial side chambers (28, 31) imports a radially asymmetric deformation characteristic to the resilient wall (23), and the fluid movement between the chambers dampers radial vibrations. The mounting device preferably has a compensation chamber (12) separate by a rigid partition (7) from the working chamber, but is fluid communication therewith by an elongate passageway (9).

Alternatively, the side chambers may be filled with air, in which case they are preferably at a radially outer edge of the resilient wall.

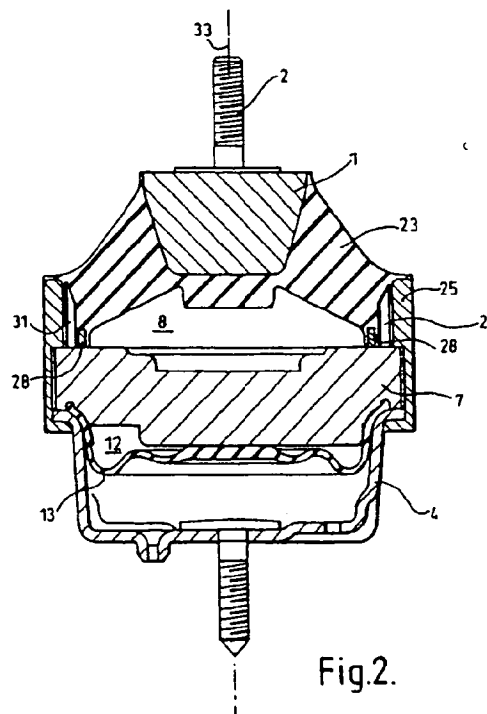


Fig.2.

Description

[0001] The present invention relates to a hydraulically damped mounting device. Such a device usually has a pair of chambers for hydraulic fluid, connected by suitable passageway, and damping is achieved due to the flow of fluid through that passageway.

[0002] EP-A-0115417 and EP-A-0172700 discussed two different types of hydraulically damped mounting devices for damping vibration between two parts of a piece of machinery, e.g. a car engine and a chassis. EP-A-0115417 disclosed various "cup and boss" type of mounting devices, in which a "boss", forming one anchor part to which one of the pieces of machinery was connected, was itself connected via a deformable (normally resilient) wall to the mouth of a "cup", which was attached to the other piece of machinery and formed another anchor part. The cup and the resilient wall then defined a working chamber for hydraulic fluid, which was connected to a compensation chamber by a passageway (usually elongate) which provided the damping orifice. The compensation chamber was separated from the working chamber by a rigid partition, and a flexible diaphragm was in direct contact with the liquid and, together with the partition formed a gas pocket.

[0003] In EP-A-0172700 the mounting devices disclosed were of the "bush" type. In this type of mounting device, the anchor part for one part of the vibrating machinery is in the form of a hollow sleeve with the other anchor part in the form of a rod or tube extending approximately centrally and coaxially of the sleeve. In EP-A-0172700 the tubular anchor part was connected to the sleeve by resilient walls, which defined one of the chambers in the sleeve. The chamber was connected via a passageway to a second chamber bounded at least in part by a bellows wall which was effectively freely deformable so that it could compensate for fluid movement through the passageway without itself resisting that fluid movement.

[0004] In the hydraulically damped mounting devices disclosed in the specifications discussed above, there was a single passageway. It is also known, from other hydraulically damped mounting devices, to provide a plurality of independent passageways linking the chambers for hydraulic fluid.

[0005] Fig. 1 of the accompanying drawings shows one example of a "cup and boss" type of mounting device, and has been disclosed in our UK patent application No. 2232430. The mounting device is for damping vibration between two parts of a structure (not shown), and has a boss 1 connected via a fixing bolt 2 to one of the parts of the structure, and the other part of the structure is connected to a generally U-shaped cup 4. A resilient spring 5 of e.g. rubber interconnects the boss 1 and the cup 4. A partition 7 is also attached to the cup 4 adjacent the ring 6, and extends across the mouth of the cup 4. Thus, a working chamber 8 is defined within the mount, bounded by the resilient spring 5 and the par-

tion 7.

[0006] The interior of the partition 7 defines a convoluted passageway 9 which is connected to the working chamber 8 via an opening 10 and is also connected via an opening 11 to a compensation chamber 12. Thus, when the boss 1 vibrates relative to the cup 4 (in the vertical direction in Fig. 1), the volume of the working chamber 8 will change, and hydraulic fluid in that working chamber 8 will be forced through the passageway 9 into, or out of, the compensation chamber 12. This fluid movement causes damping. The volume of the compensation chamber 12 needs to change in response to such fluid movement, and therefore the compensation chamber 12 is bounded by a flexible wall 13.

[0007] In use, the force received by the mounting device is principally parallel to the fixing bolt 1, and this direction defines an axis of the boss 1.

[0008] The above structure is generally similar to that described in EP-A-0115417, and the manner of operation is similar. In EP-A-0115417, the partition supported a diaphragm which acted as a boundary between fluid in the working chamber and a gas pocket. In the arrangement shown as Fig. 1, there is an annular diaphragm 50 which is convoluted. That diaphragm 50 is held on the partition 7 by an upper snubber plate 22, that snubber plate 22 is held in place by a ring 40, which is clamped to the partition 7 and to the cup 4, by a clamping ring 41. The resilient spring 5 is also connected to the ring 40. The upper snubber plate 22 has openings 21 which permits fluid in the working chamber 8 to contact the diaphragm 50.

[0009] In the arrangement shown in Fig. 1, the passageway 9 is in the form of a spiral, and the internal dimensions of that spiral are uniform.

[0010] In its most general terms the present invention proposes that in a hydraulically damped mounting device for supporting an axial load, a resilient wall which connects two anchor bodies and surrounds an axis of the mounting device, is formed without circumferential symmetry, for example by including one or more cavities.

[0011] In accordance with the present invention, a hydraulically damped mounting device for supporting an axial load may include a boss defining an axis and a second anchor body axially spaced from the boss, a resilient wall extending circumferentially around the boss and from the boss to the second anchor body, thereby defining a central chamber bounded by the boss, second anchor body and resilient wall, the resilient wall including one or more side chambers (i.e. cavities), and either or both of (i) the central chamber or (ii) at least one of the side chambers, being filled with hydraulic fluid.

[0012] The number and locations of the side chambers are selected to produce a selected stiffness characteristic. For example, two side chambers may be provided in opposed locations on either side of the axis. The side chambers may then be symmetric with respect

to reflections in a first plane including the centre of the side chambers and the axis of the boss, and in a second plane perpendicular to the first and including the axis of the boss, although the stiffness of the mounting device with is different in the two planes.

[0013] Furthermore, the resilient wall is usually generally conical in shape, so that even a purely vertical vibration has a radial-component. Therefore, it is possible that the chambers may provide vertical damping.

[0014] In a first form of hydraulic mounting device according to the invention, the second anchor body is the cup of a "cup and boss" mounting device and the central chamber is filled with hydraulic fluid which acts as a working chamber. However, one or more of the side chambers are not in communication with the working chamber, but are formed with apertures opening in the surface position of the resilient wall located outwardly of the seal between the resilient wall and the mouth of the cup, so that the side chambers are filled with air. The side chambers thus provide weakened portions in the resilient wall, reducing the stiffness (primarily the radial stiffness) of the mounting to deformations which include compression of the side chambers.

[0015] In a second form of hydraulic mounting device according to the invention also, the second anchor body is the cup of a "cup and boss" mounting device, but in this case there are at least two side chambers which are interconnected and both the side chambers and the central chamber are filled with a hydraulic fluid. Preferably there is also communicating with the working chamber via a vent conduit.

[0016] In this case, the connecting channel(s) connecting the two side chambers, could consist of a further cavity of reduced width extending circumferentially around the axis of the boss between the side chambers. Alternatively, the conducting channel could be formed in a ring surrounding the radially outward surface of the resilient wall. The resilient wall may for example be formed with a varying radius about the axis of the boss, being of greater radius in first portions including side chamber(s) and reduced radius in second portions without side chambers, the ring extending radially inwardly to a greater extent in the second portions and including the conduction channel(s).

[0017] Fluid flowing between the chambers and/or between the chambers and the working chamber permits the walls of the side chambers to flex upon deformation of the resilient wall, and thus the stiffness of the resilient wall is reduced in a plane intersecting with the side chambers.

[0018] The dimensions of the fluid vent(s) and channel(s) can be selected to produce a desired damping characteristic of radial vibrations in one or both of the mirror planes. This effect could be used either to dampen vibrations over a particular frequency range, or alternatively to provide a reduced dynamic radial stiffness over a given frequency range.

[0019] For example, the size of the vent may be se-

lected to be narrow enough to effectively prevent fluid flow except on long time scales.

[0020] Control elements may be provided within the mounting to vary the radial damping under selected vehicle conditions by altering the dimensions of the vent(s) and/or channel(s). For example, an electro-mechanical or pneumatic control means may be included to increase or reduce the size of the vent(s).

[0021] In each of the first and second forms of hydraulic mounting device, the central chamber, which is filled with hydraulic fluid, is preferably the working chamber of a cup-and-boss hydraulic mounting device, and communicates with a further chamber, such as a compensation chamber, for example by a further passageway in the cup.

[0022] However, in a third form of hydraulic mounting device according to the invention, the central chamber is not filled with hydraulic fluid, but at least one side chamber is. Preferably, there are two side chambers, connecting by a cavity of reduced width extending circumferentially around the axis of the boss between the side chambers. In this case, the axial load-bearing capacity of the bearing capacity of the bearing results primarily from the resilient wall itself. However, motion of the hydraulic fluid between the side chambers damps relative motion of the boss and support body. Such a flow of hydraulic fluid could for example be caused by a radial motion of the boss, so that radial motions of the boss are damped by this mounting device.

[0023] If there are two side chambers provided on opposite sides of the central chamber, then radial motions of the boss in the plane of the chambers are damped to a greater degree than radial displacements perpendicular to this plane. However, would also be possible to provide more side chambers, for example four chambers located equally spaced circumferentially around the central chamber, so that the degree of damping was made less dependent upon the direction of axial displacement.

[0024] Embodiments to the present invention will now be described in detail, by way of example, with reference to the accompanying drawings. In which;

Fig. 1 is a sectional view through a known hydraulically damped mounting device, and has been discussed above;

Fig. 2 is a sectional view of a first hydraulically damped mounting device according to the invention;

Fig. 3 is a sectional underside view of the hydraulically damped mounting device of Fig. 2;

Fig. 4 is a sectional view of the hydraulically damped mounting device of Fig. 2 in plane X-X on Fig. 3;

Fig. 5 is a sectional view of a second hydraulically damped mounting device according to the invention in a plane including the axis; and

Fig. 6 is a sectional view of the hydraulically

damped mounting device of Fig. 5 in a perpendicular plane including the axis.

Fig. 2 shows a cross-sectional view of a first embodiment of the invention, in which the resilient wall 23 surrounding the working chamber 8 is formed with two side chambers 27, 31 in opposing locations.

[0025] As can be seen from this figure, the resilient wall 23 extends from boss 1 to partition 7 and a ring 25. It includes a pair of side chambers 27, 31 on either side of the axis 33 of the boss 1, and slightly radially inward of the main part of the ring 25. The ring 25 also has an inner part 28 radially inward of the side chambers 27, 31. In the cross-section of Fig. 2, this inner part 28 appears separate from the rest of the ring 25. In fact, this is not the case, and the inner part 28 is joined to the rest of the ring 25 at other circumferential positions, as is shown in Fig. 3.

[0026] Fig. 3 is a cross-sectional view of the embodiment of Fig. 2 in a plane perpendicular to the axis 33. The view of Fig. 2 is the plane y-y shown in Fig. 3. The two side chambers are connected to each other by a conducting channel 26 which extends circumferentially around the working chamber. Each conducting channel is connected to the working chamber 8 via a vent 29.

[0027] Fig. 4 shows a cross-section of the embodiment of Fig. 2 in the plane X-X marked on Fig. 3. This illustrates that in this plane the ring 25 extends radially inwardly to a greater extent, and includes the conducting channel 26 which connects the side chambers 27, 31. A vent 29 connects the connecting channel 26 to the working chamber 8.

[0028] The side chambers 27, 31 are thus supported by hydraulic fluid contained within them. Deformation of the mounting causes deformation of the chambers to an extent determined by the flow of hydraulic fluid between the side chambers and/or into or out of working chamber 8, which is also filled with hydraulic fluid. Thus, by appropriate selection of the dimensions of the conducting channel 26 and the vent 29, the stiffness of the hydraulic mounting can be selected optimally, for example at a selected frequency range.

[0029] Although in the embodiment described above the side chambers are filled with hydraulic fluid and in communication with the working chamber 8, in other embodiments of the present invention the side chambers are formed radially outwardly of the seal between the resilient wall and the partition. For example, the sealing element 28 may extend around the entire periphery of the working chamber. In this case the side chambers may be filled with air, and optionally open to the environment of the mounting device via an aperture.

[0030] Furthermore, the asymmetry of the resilient wall may also be effected by means other than, or in addition to, the side chambers, such as by variations in the thickness of the resilient walls, for example proximate

to the partition, and/or by the inclusion of additional elements with elasticity higher or lower than that of the material composing the resilient wall.

[0031] For simplicity, the partition of the embodiment described is shown as being solid and rigid. While it is indeed possible for the partition to be solid and rigid, it is preferable that the partition includes one or more passageways (usually just one) linking the working chamber to a compensation chamber, and it may itself be resiliently deformable and/or include separate moving elements.

[0032] A second embodiment of the invention will now be described with reference to Figs. 5 and 6 which correspond respectively to Figs. 2 and 4 used to illustrate the first embodiment.

[0033] In the second embodiment, a resilient wall 123 extends radially outwards from a boss 1 to a second anchor body 117. A central chamber 18, which is filled with air, is defined between them. The resilient wall includes two side chambers 127, 131 formed on opposite sides of the central chamber 18. These side chambers are filled with hydraulic fluid.

[0034] As shown in Fig. 6, a passageway 126 connects the side chambers 127, 131. In contrast to the first embodiment, no vent is formed between the passageway 126 and the central chamber 18.

[0035] In this embodiment, displacements of the boss 2 to either side in Fig. 5 drives hydraulic fluid from one of the side chambers 127, 131 and into the other side chamber through the passageway 126. Thus, the motions of the boss in this direction are damped. Vertical motions of the boss 1 relative to the second anchor body 117 are resiliently resisted by the resilient wall 123.

Claims

1. A hydraulically damped mounting device for supporting a load, comprising first and second anchor bodies (1,7) vibratable relative to each other in an axial direction, a resilient wall (23) interconnecting the first and second anchor bodies and supporting the first anchor body (1) axially above the second anchor body(7), a working chamber (8) for hydraulic fluid axially between the first and second anchor bodies, the working chamber (8) being partially bound by the resilient wall (27);
characterised in that:
the resilient wall includes at least one side chamber (27, 31) arranged to impart a radially asymmetric deformation characteristic to said resilient wall (27).
2. A hydraulically damped mounting device according to claim 1, having two said side chambers (27,31) on opposite sides of the axis.
3. A hydraulically damped mounting device according

- to claim 1 or claim 2, wherein the side chamber(s) are filled with air.
4. A hydraulically damped mounting device according to claim 3, where in said side chamber(s) are at a radially outer edge of the resilient wall (23). 5
 5. A hydraulically damped mounting device according to claim 1 or claim 2, wherein the side chamber(s) (27,31) are filled with hydraulic fluid. 10
 6. A hydraulically damped mounting device according to claim 5, wherein the side chamber(s) (27, 31) communicate with the working chamber. 15
 7. A hydraulically damped mounting device according to claim 5 or claim 6 as dependent on claim 3, wherein the side chambers (27,31) are interconnected by a connecting channel (26). 20
 8. A hydraulically damped mounting device according to claim 7 where the connecting channel (26) is within the resilient wall (28). 25
 9. A hydraulically damped mounting device according to claim 7, wherein the connecting channel is in a ring surrounding the resilient wall. 30
 10. A hydraulically damped mounting device according to any one of the preceding claims, wherein the resilient wall (28) has varying radius. 35
 11. A hydraulically damped mounting device according to any one of the preceding claims, wherein the first anchor body (1) is in the form of a boss, the resilient wall (28) extends around the boss and from the boss to the second anchor body(7), and the working chamber (8) is defined by the boss (1), second anchor body (7) and resilient wall (28). 40
 12. A hydraulically damped mounting device according to any one of the preceding claims, further including a compensation chamber (12) for the hydraulic fluid, and a passageway (9) interconnecting the working (8) and compensation (12) chambers. 45
 13. A hydraulically damped mounting device according to claim 12, wherein the passageway is in a rigid partition forming part of the second anchor parts (7) and partially bounding the working chamber. 50

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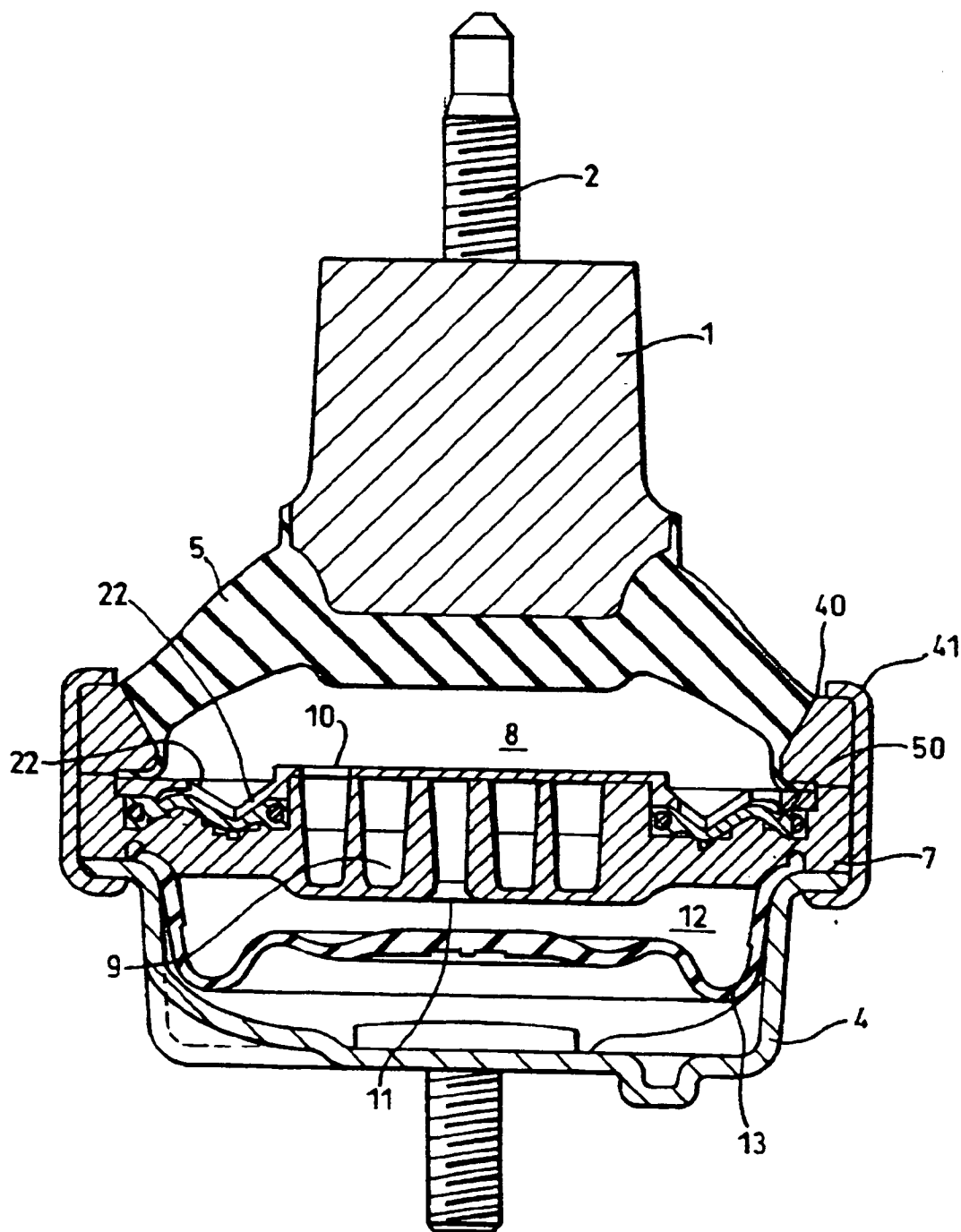
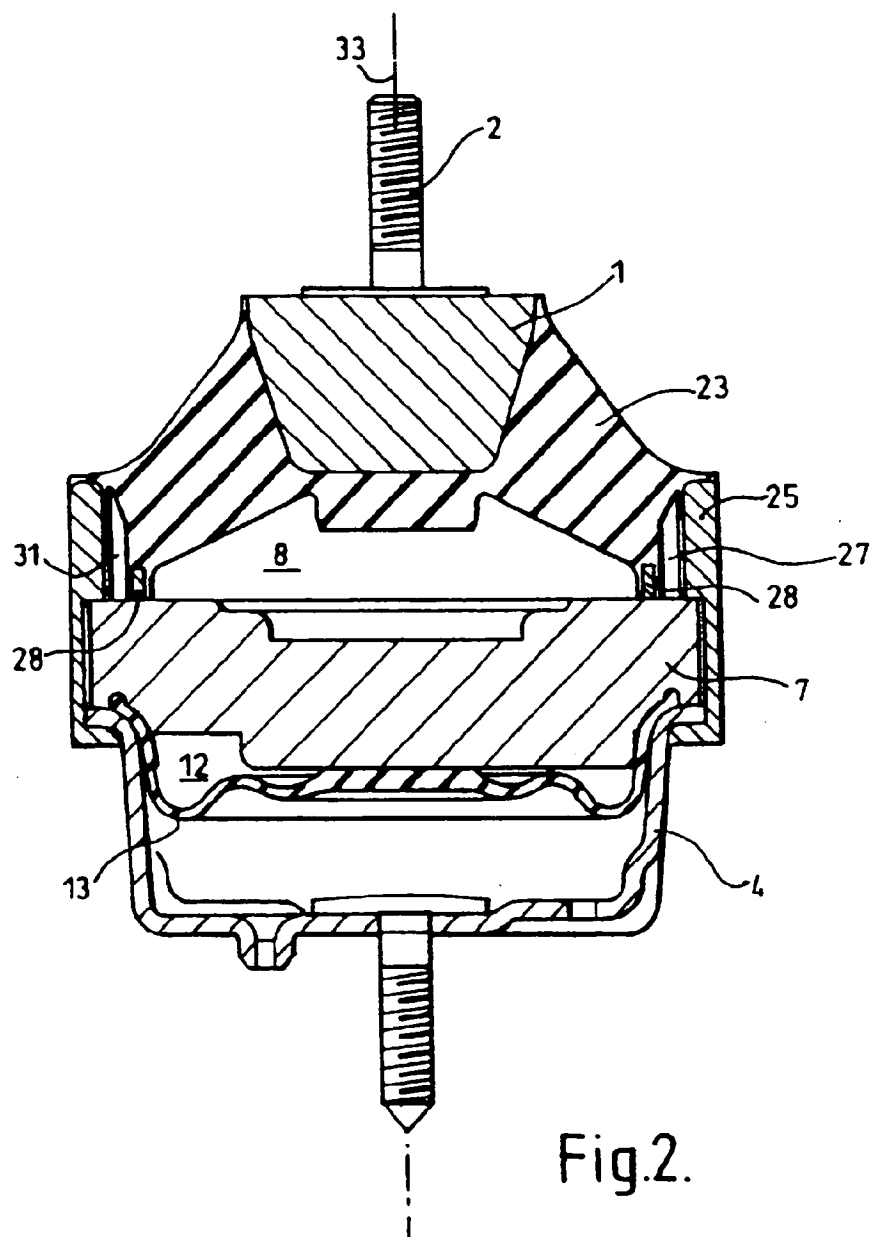


Fig.1.



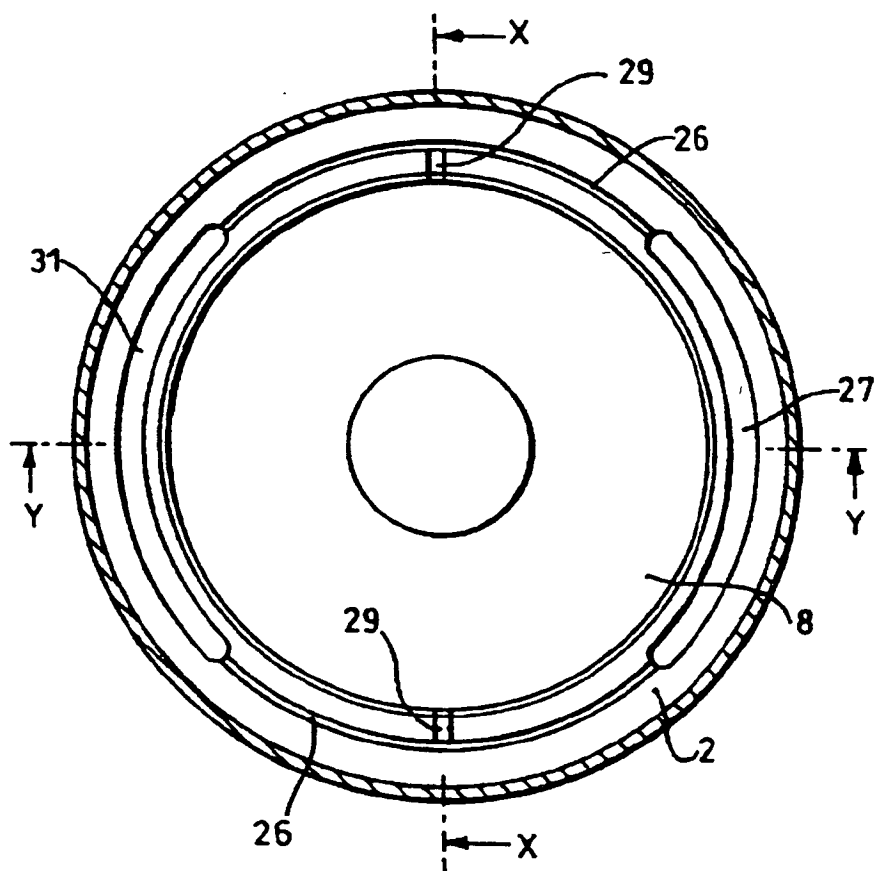


Fig.3.

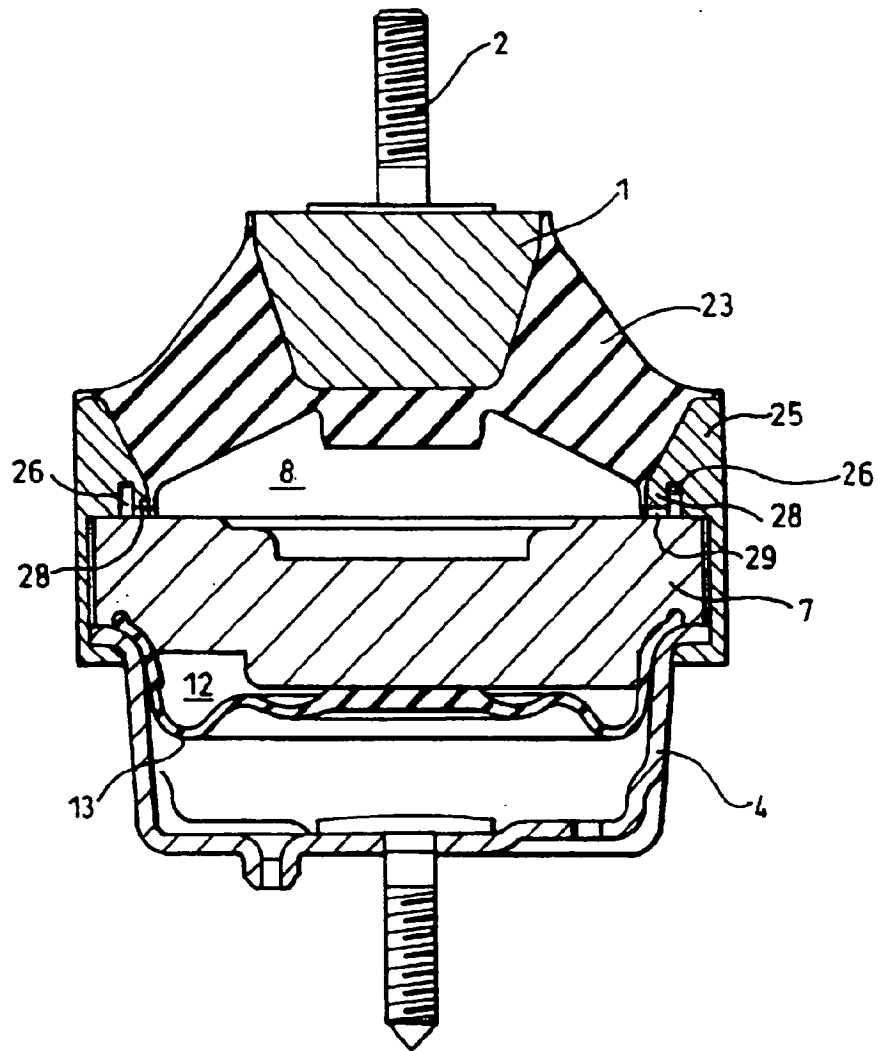


Fig.4.

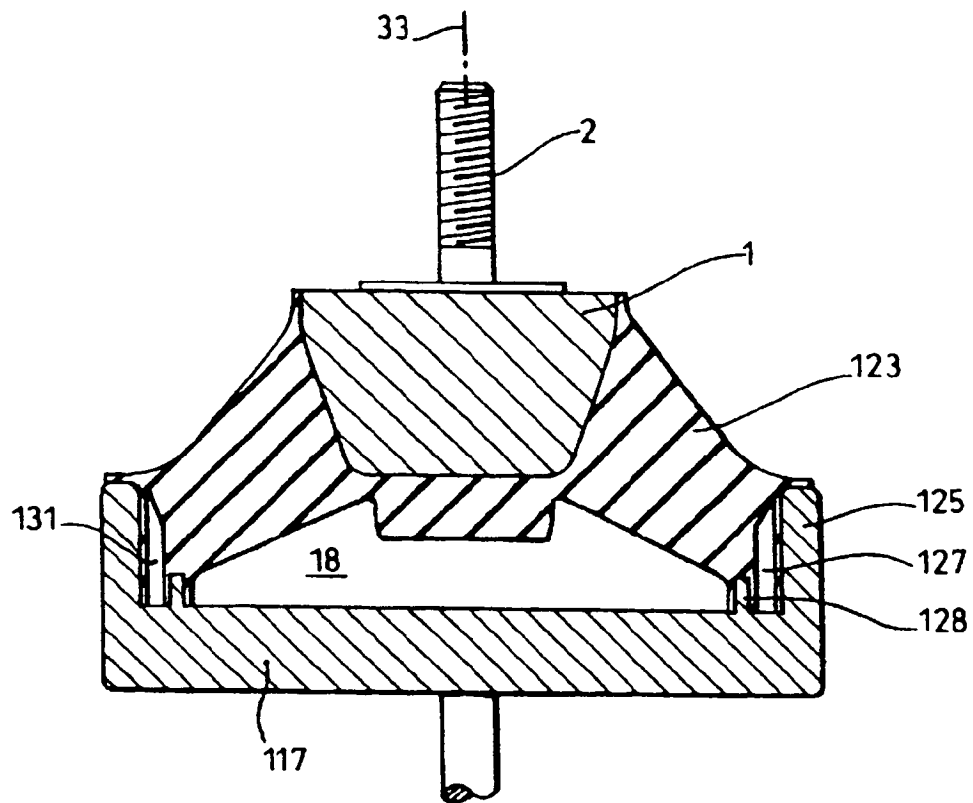


Fig.5.

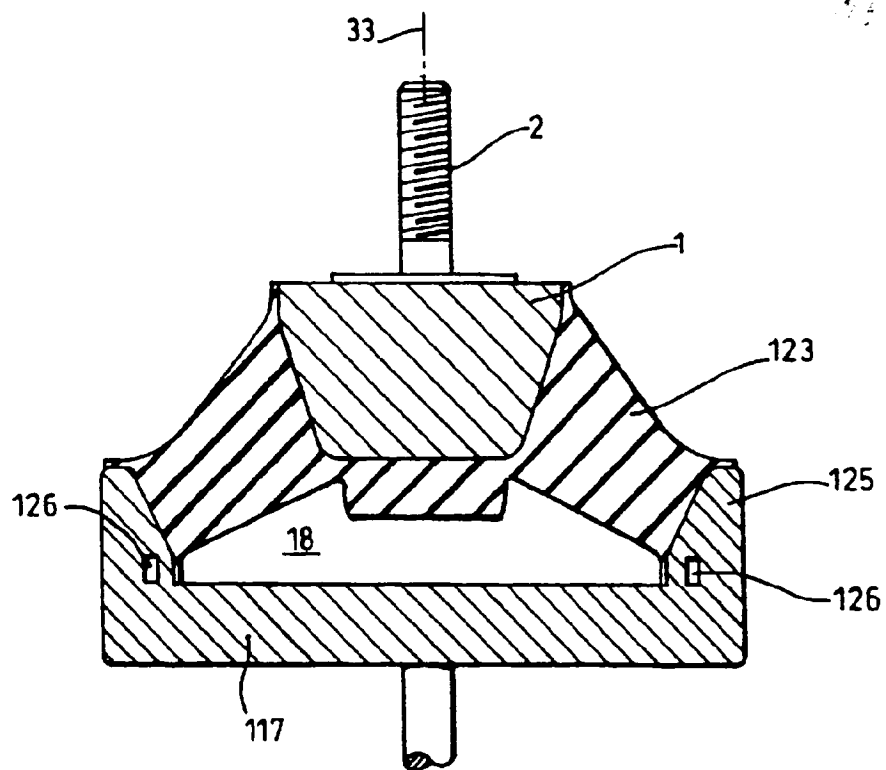
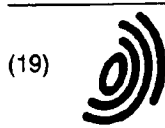


Fig.6.

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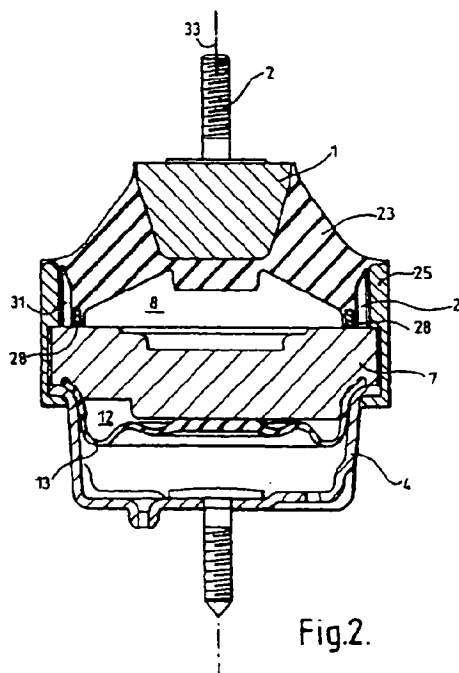
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EUROPEAN SEARCH REPORT

Application Number
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Place of search THE HAGUE		Date of completion of the search 12 December 2001	Examiner Beaumont, A
<p>CATEGORY OF CITED DOCUMENTS</p> <p>X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document</p> <p>T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document</p>			

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**ANNEX TO THE EUROPEAN SEARCH REPORT
ON EUROPEAN PATENT APPLICATION NO.**

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